

立式轮盘变幅循环旋转试验器

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为了较精确地预示轮盘低循环疲劳寿命, 我们将轮盘超转、破裂试验器改造成应用微机自动控制的变幅循环旋转试验器, 成功地完成了轮盘的变幅低循环疲劳旋转试验, 给出了轮盘幅板偏心孔的裂纹起始寿命和扩展寿命。

1. 变幅循环试验系统 立式轮盘变幅循环旋转试验器由原轮盘超转、破裂试验器经改造后配备转速自动控制微机系统组成(图1)。设备改造的关键是变幅低循环疲劳微机控制系统。图2表示了试验器经改造后的转速循环自动/手动控制系统。通过计算机编程可实现模拟任意飞行起落循环转速波形。图3所示为典型的截击机的飞行循环。通过 TP801 键盘输入如图3所示的各峰、谷转速数值值和相应的负载时间, 将转换开关拨到机控挡, 启动计算机, 通过D/A转换器按程序设定的加、减速要求自动控制可控硅调速系统, 即可实现其转速波形循环。计算机通过磁电传感器(见图2)反馈回来的转速信号进行实时快速计算, 自动校正预定转速与实测转速之间的误差, 以满足试验对稳速精度和循环重复精度的要求。为了提高工作可靠性, 防止环境干扰, 控制程序编定后, 经固化可保证长期稳定工作。为了避免意外故障对试验转子和设备造成损失, 在系统上设计了超转速、超加速和

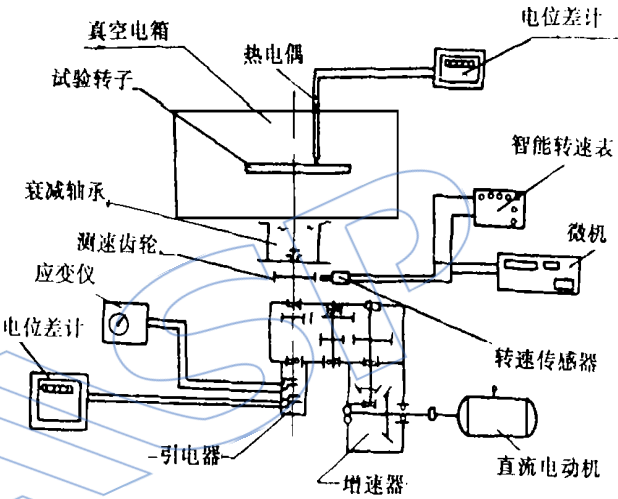


图1 试验器结构示意图

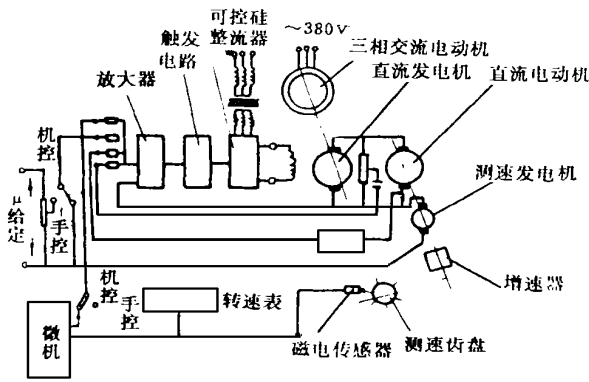


图2 微机控制系统框图

超电流保护装置。一旦有故障，系统就会报警并自动停车。新研制的微机转速循环控制器还还有仪表功能，在上安装了预置转速和实测转速显示器、循环疲劳计数器以及指示转速给定值的直流电压表等。

设备经过改造后，对三种不同惯量转子进行了严格的验收试车，每种转子都按图 4 所示的波形进行试验，有关参数见表 1。另外，还对一个实际型号第九级压气机转子和一个模型盘分别进行了等幅低循环疲劳试验和主、子复合变幅低循环疲劳试验，二个转子累计疲劳试验 2000 多小时。

设备经改造后的综合性能为: 设备驱动功率 125K W (直流电动机 0 ~ 1500r/min); 试验转子最大直径 $\Phi 1m$; 试件最大重量 300kg; 转速范围 0 ~ 25000r/min; 最高试验温度 800 ± 5 ; 试验腔真空度 400 ~ 667Pa; 稳速精度 $\pm 0.1\% \sim \pm 0.3\%$ ($n=25000 \sim 4000r/min$); 循环重复精度 $\pm 0.1\% \sim \pm 0.3\%$ ($n=25000 \sim 4000r/min$); $\pm 0.3\% \sim \pm 2\%$ ($n=10000 \sim 1000r/min$); 转速波形为梯形、三角形、任意波形; 可试项目有应力温度测量、超转、超温、破裂、叶片包容试验; 等幅、主、子复合及随机低循环疲劳试验。瞬时加、减速的设备振动 $3g$ (最大加速电流 $\leq 500A$)，稳速运行时 $< 2g$ 。

表 1 三种考核转子试验转速

项目	试验件 转 速	实际型号转子 $\Phi 488, 50kg$	标准模拟转子 $\Phi 180, 35kg$	GH 901 模型盘 $\Phi 420, 25kg$
稳速	上限/下限	1200/4000	22000/6000	16000/4000
等幅	上限/下限	12000/1600	20000/1600	16000/3000
复合	上限/中限/下限	12000/9000/1600	24000/6000/1600	16000/9000/3000
随机	1-1'	12000-9000	22000-1000	16000-1000
	2-2'	11000-6000	15000-2000	15000-2500
	3-3'	10000-8000	13000-8000	14000-9000
	4-4'	12000-1600	18000-1600	16000-3000

复合低循环疲劳试验任务。累计循环块数 3332 块，总运行时数为 667 小时。微机控制系统达到了预定的试验要求。循环过程中转子即使在临界状态下工作，振动也很小。

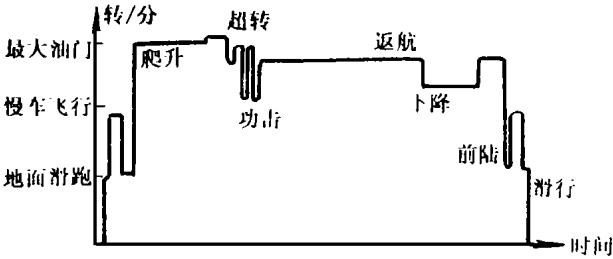


图 3 发动机的飞行循环

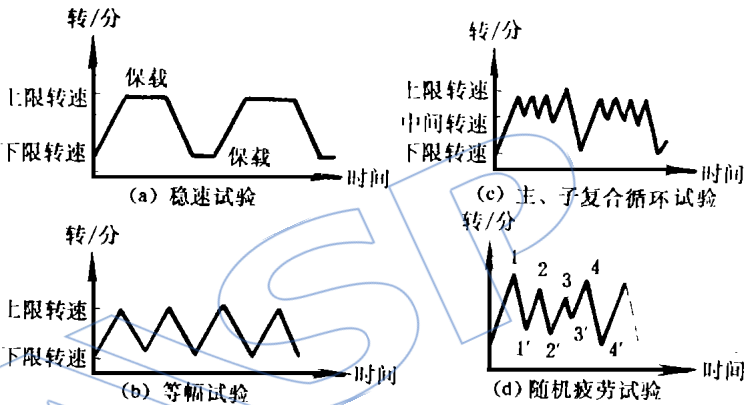


图 4 验收试验波形

2. 实际应用

(1) 完成了 50kg 级 $\Phi 488$ 某第九级压气机盘 1600 ~ 11850 转/分之间的等幅循环试验，累计循环 25000 次，历时 1200 多小时。(2) 完成了一个 25kg 级 $\Phi 422$ 的 GH 901 模型盘主、子

NATURAL VIBRATION ANALYSIS OF TURBOMACHINERY DISK WITH SMALL MISTUNED BLADES

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ABSTRACT

A numerical approach is developed for natural vibration analysis of turbomachinery disk with small mistuning blades by using modal synthesis technique. Due to lack of rotational periodicity, the group theory method for the tuned bladed-disk assemblies is not applicable to the mistuned ones. According to the matrix perturbation theory, the modes used for synthesis in this paper are taken from a tuned bladed-disk corresponding to the small mistuning one. This new technique keeps the advantages of the group theory method and makes the final eigenvalue problem of small size. A computer program has been written to analyse the natural vibration characteristics of turbomachinery disk with small mistuning blades. The computational results prove that the method is very efficient.

TECHNICAL NOTES

VERTICAL ROTATING TESTER WITH VARIABLE-AMPLITUDE CYCLE FOR GAS TURBINE DISKS

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ABSTRACT

A microcomputer-controlled high-speed high-temperature rotating tester with variable-amplitude cycle has been constructed and operated to estimate the LCF life of gas-turbine disk. So far the tester has worked about 2500 hours (total cycle index is about 50000), without essential troubles. The data of the variable-amplitude LCF life for several disks have been obtained. These data are of high quality. It is testified that the test equipment is sufficient for performance research of rotating disks with diameter up to 1m and weight up to 300kg. The disk is mounted vertically on flexible shaft; an D. C. motor (125kw) drives the disk up to 25000rpm. The heater can raise the temperature of the disk up to 800 ± 5 . The average speed error for a run is less than 0.1%; the maximum instantaneous error is about 0.5%. A vacuum pump is used to sustain about 400 to 667Pa in the chamber of the tester.

A LOW CYCLE FATIGUE LIFE PREDICTION FOR DISK GROOVE BOTTOMS

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ABSTRACT

According to the theory of local strain fatigue, the fatigue experiments for the 30CrMnSiA disk have been completed on a multichannel radial loading fatigue rig ("Ferris" wheel). A good number of the crack propagation curves of the groove bottoms in the disk have been obtained. The calculations show that the median LCF life to